

^{11. (}a) A V-belt drive is to transmit 40KW in a heavy duty saw mill which works (14) 1 in two shifts of 8 hours each. The speed of motor shaft is 1440 rpm with the approximate speed reduction of 3 in the machine shaft. Design the drive.

- (b) Design a chain drive to run a compressor from running at 970 rpm, the compressor speed bein operates 16 hr per day. The center distance 500mm. The chain tension can be adjusted by set operated by set of the compression of the set of th
- **12. (a)** Design a spur gear pair to transmit 5kW at 144 to an air compressor running at 720rpm. Take Both the gears are made of the C45 steel.

(OR

- (b) A pair of helical gear subjected to moderate shows at 1500 rpm of the pinion. The speed reduced angle is 20°. The service is continuous, and the normal plane. For the gear life of 10,000 holes.
- 13. (a) Design a pair of right-angled bevel gears to the another gear to run at 250 rpm. Not less that either gear. The pressure angle is 20⁰. Assume

(OR)

- (b) Design a worm gear drive to transmit a pow speed is 1440 rpm and the speed of the wheel have a minimum efficiency of 80% and above. the worm and the wheel.
- 14. (a) Design a 9-speed gear box for drilling mach Minimum speed is 100 rpm and the recomment the speed diagram, kinematic diagram.

(**OR**)

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- (b) Design the layout of a 12 speed gear box application having an output of speeds ran Choose standard step ratio and construct the diagram.
- 15. (a) A single plate cutch transmits 25kW at 900 pressure between the plates is 85 N/mm². The side of the plates are effective and the coord determine (i) the inner diameter of the plate. the clutch. Assume the theory of uniform wear (OR)
 - (b) A 360 mm radius Brake drum contacts a single b and resists a torque of 250 Nm at 500 rpm.
 0.3. Determine (i) The normal reaction on the applied at the lever end for counterclockwise

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(OR)

om an 11 KW electric motor ing 330 rpm. The compressor ce should be approximately shifting the motor on slides.	(14)	1	3
40 rpm from an electric motor e working life as 10,000 hrs.	(14)	2	3
) hock loading is to transmit 20 uction ratio is 4 and the helix the teeth are 20° full depth in ours, design the gear drive.	(14)	2	3
ransmit 15 kW at 750 rpm to n 20 teeth are to be used on a gear life of 12000 hrs.	(14)	3	3
) wer of 22.5 KW. The worm l is 60 rpm. The drive should . Select suitable materials for	(14)	3	3
ine for the following inputs. ended step ratio is 1.25. Draw	(14)	4	3
) for a crushing machine tool ging from 100 to 355 rpm. speed diagram and kinematic	(14)	4	3
rpm. The max. intensity of he ratio of radii is 1.25. Both efficient of friction is 0.25. (ii)the axial force to engage	(14)	5	3
) le shoe as shown in Figure15 The co-efficient of friction is he shoe, (ii) The force to be rotation of the drum if <i>e</i> is 0,	(14)	5	3

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(iii) The force to be applied at the lever end for clockwise rotation of the drum if e is 42 mm.



Figure15 b

PART- C (1x 10=10Marks)

(Q.No.16 is compulsory)

Marks	CO	RBT
		LEVEL
(10)	4	3

16. For construction of slotting machine tool, a six-speed gear box is required to (10) provide output speeds in the range of 125 to 400 rpm, as per the need of end user. The Renard series of R(10) is recommended by design team, to arrive at six levels of speeds. To meet the functional requirement, the committed power for transmission from motor is of 5 kW at a speed of 710 rpm. Design and detail the speed diagram, kinematics diagram and number of teeth required.

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